EXPANSION COMPENSATORS & HOUSED EXPANSION JOINTS EC & HEJ

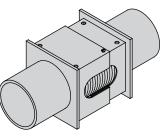
We are all interested in product development, but it is often difficult to trace. Let us share what we have learned.

When applying internally pressurized expansion joints, the designer has to be concerned with extension, compression, angular and torsional motion. Like automobile tires, rubber expansion joints are thick-skinned, forgiving creatures that tolerate abuse. Stainless steel is very reliable too, but only if expertly designed, properly anchored and guided. All of us have taken a strip of metal and bent it back and forth until it cracked and snapped. Multiple corrugations designed to low stresses eliminate the problem.

An anchor close to an expansion joint stabilizes one side. Then two guides, one 4 diameters from the joint and the other a minimum of 14 diameters from the first, lead the piping straight in and out. If the anchors are both up and down-stream, four guides are required, two on each side. Improper anchoring or guiding leads to failure. If major movements are required, it may become necessary to increase the outside diameter to prevent buckling. This increases both thrust and cost.

There is always the worry of personal injury from hot liquid or steam, even though the evolution from steel to galvanized steel, copper, bronze and finally stainless steel, has increased service life and operating pressures.

Some designer tried to solve some of these internally pressurized problems by telescoping two square housings around the joint. It reduced the rotational problem, but without the two external guides, the possibility of angular failure is always there.

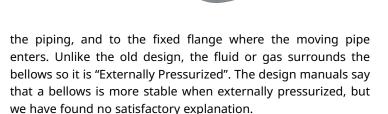


Some years later, there was a major change in concept. The industry turned to a completely housed construction. In this design, the bellows is welded to a sliding flange on the end of

2" (51mm) COMPENSATOR WITH SINGLE BELLOWS WITH SINGLE BELLOWS Siding Internal Flange Flange Siding Pressurized

Externally Pressurized Expansion Joints

Internally Pressurized Expansion Joints

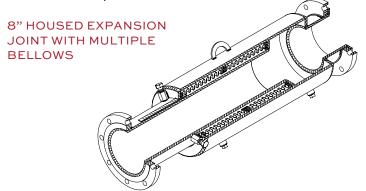


Self guiding is provided by the pipe passing through the opening on the one end and the small clearance of the moving flange inside the outer housing. Should there be a bellows failure, the escape is only through the lower clearance and back along the pipe, rather than in an unpredictable direction. The housing is a great safety feature, particularly in steam lines.

> Path of Fluid or Steam In Failure

For whatever reason, the entire industry refers to 2" (51mm) movement designs as Expansion Compensators. They are shipped preset to allow for inward travel only in hot lines. Should there be a dual temperature situation, there is no engineering reason the compensator cannot be moved in 3/4" (19mm) before installation, so it would accommodate 11/4" (32mm) pipe expansion and 3/4" (19mm) contraction or some other setting.

When the rated inward motion is increased to 4" (102mm) or more, the industry name changes to Housed Expansion Joints, which is a much better description. "Expansion Compensator" is a function, not a product.



Oddly enough, some people still describe the 2" (51mm) movement Expansion Compensators as internally pressurized. They are not. Only the old design on the previous page is internally pressurized.

We had no illusions about coming up with a very different approach, but it is never our way to introduce a product without improvement. So just as we did with the straight hoses and Vees, we purchased twenty or thirty compensators and housed expansion joints from various manufacturers to see what was going on.

Perhaps the original engineer designed a whole range of consistent products. However, the more samples we looked at, the more confusing it became. There was no consistency. When people copy products to cheapen them or never completely understand the original design, the copy often suffers.

Virtually all manufacturers advertise 2" (51mm) movement multi-ply bellows. Many companies just buy and resell without writing specs or testing. When their suppliers deliver single ply, the resale company would not know. There is no great harm, as the only reason to go to multiple plies is to increase pressure ratings without using a single thicker wall. A single thickness with the same corrugations and movements would suffer higher stresses, a shorter service life and the bellows would have a higher spring rate even though low spring rates are relatively unimportant. To maintain the same service life, the bellows would have to be longer to reduce movement per corrugation, and in addition to space considerations, more costly designs to manufacture.

Moving on to 4" (102mm) and 8" (203mm), most manufacturers use 2 ply, and some 3. The number is not particularly important. However, with greater movement, it becomes more difficult to maintain stability. Any stretched bellows is stable. However, pipelines do not only expand, they must cool and contract. When the motion is reversed and the bellows compressed, it can become unstable and buckle. Most manufacturers design to the maximum stable length and weld them to guide rings between sections to prevent buckling of a longer column. Our own designs vary from product to product, just depending on what works best. Since people like to know what they are buying, we note the number of bellows and plies for all products at this writing. Most companies do not.

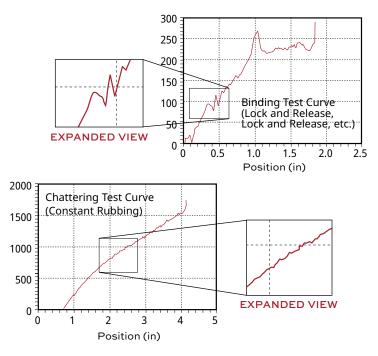
We found that to prevent buckling, some manufacturers had their bellows rubbing against the inside pipe or the outer shell. It would seem they must wear through and leak earlier, so tight fits are a poor way to go. (See bronze bellows photo below)

Others went to huge outer housings as compared to the inner pipe, so their products were clumsy (below). In our own designs, we have succeeded in providing good working clearances, both between the inside of the bellows and the moving pipe as well as the outer shell. We note clearances on all product drawings. Competitors do not. (See pages 10.78 - 10.84)



In checking spring rates, we found that some competitors are so concerned with alignment, that clearances between the moving pipe and the entry collar, as well as the moving inner flange and the outer shell, were so tight that binding was a serious problem. We are publishing a few test curves to show this condition.

COMPETITOR'S SPRING RATE CURVES

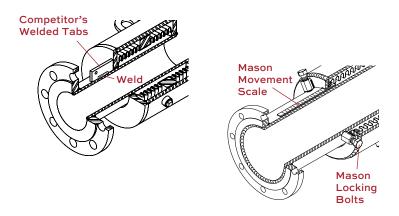


Since there is no need for so small a tolerance, our clearances are roomier for smoother travel, as shown by the Mason test curve below. When expansion joints bind, it increases the anchorage loads. Binding intermediate rings stretch bellows unevenly and can cause failure.

> 1400 1200 1000 (dl) beo-800 600 400 Smooth Action without Binding 200 or Chatter 0 0 2 3 4 5 Position (in)

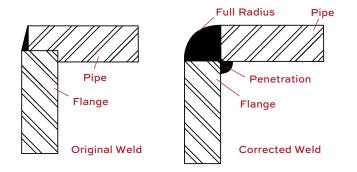
TYPICAL 4" TRAVEL MASON SPRING RATE CURVE

Another improvement is our introducing locking bolts to maintain the installed position. All of the other manufacturers use welded tabs. The tabs are knocked out after the piping is fixed to the expansion joint. Sometimes they are flimsy and fall off. More importantly, the side welds are often within the movement of the pipe entering the housing. The weld has to tear its way through until it clears. Not the smooth motion we would all like to have.



Perhaps our most interesting contribution is the introduction of a scale to confirm the starting position is zero for the average installation. Many of the stop-tabbed samples we checked were not. When the joint is in service, the scale indicates how much the pipe has expanded and how much the bellows have been extended. In a hot and cold situation, you can loosen the restraining bolts, and push the bellows to the desired starting position to allow for a line cooling down or a seasonal change in temperature. Reading the scale makes it easy. Re-lock the bolts and continue with the installation. (NOTE: the 2" (51mm) movement expansion compensators have a 0, 1" (25mm) and 2" (51mm) reference only.) If for whatever reason you have to remove an expansion joint temporarily, you can retighten the locking bolts, remove it, do the maintenance work and reinstall the expansion joint without fighting it into position as you would have to with any other device. Loosen the bolts and it is back in service. Welded tabs would have been long gone. While any good mechanic can find a way, this is much easier.

Shortly after we sold our first housed expansion joints, we had a serious failure even before they were put into service. One of our contractors was prefabbing as much as possible to minimize time on the jobsite. Rather than installing the housed expansion joint between pipe ends in the field, he welded long extensions to the pipe on both sides. In normal rough handling, one of the modified joints was dropped. The cantilever of the long welded pipe snapped the flange off the fixed end. When we got it back in the shop, we found the method we were using in machining the flange to the I.D. of the housing left a small lip around the pipe perimeter as sketched. This undersized weld looked great and would have held the pressure, but not impact.



Here again, we checked competitor's products, and found our original technique was typical. We re-examined all of our welds and changed every one to good piping practice with welds as sketched. Maybe this is overkill, but neither you nor we will ever suffer a weld failure again and that's good to know.

Earlier, we suggested the bellows spring rate was insignificant. The reason is the thrust from a housed expansion joint is equal to the area as calculated to about the center of the bellows, multiplied by the pressure in the pipe line. Typically a 4" pipe bellows might have a 61/4" diameter to the center of the corrugation. At 225 psi, the thrust force is a nominal 6900 lb. The spring rate of our 4" travel HEJ is 300 lb per inch. The 8" HEJ is 150 lb per inch. Therefore, with either joint, the total bellows resistance at 4" or 8" travel is 1200 lb. This increases the anchor load to 6900 plus 1200 lb or 8100 lb total. 1200/8100 = 15%. If a competitive product were half as stiff, the anchorage requirement would drop to 7500 lb. If twice as stiff, it would increase to 9300 lb. In the real world, when designing an anchor, all these numbers are in the same order of magnitude. The spring force is relatively insignificant, except for an extremely unusual installation, where there might be concern for buckling of copper pipe or something of that nature.

SAFETY FACTOR

Our last worry was the question of safety factor.

A flexible hose or internally pressurized expansion joint fails because the walls fail in tension beyond a given pressure. Since all housed expansion joints are pressurized externally, that is not the phenomena. The pressure on the outside eventually forces the corrugations to squirm and become distorted until they collapse completely, as shown by one of our test photographs to the right.

According to the standards established by The Stainless Steel Expansion Joint Institute, an expansion joint is considered safe when this collapse occurs at 2.25 times the working pressure. That means the safety factor is only 2.25. Most manufacturers do not publish their collapse ratios, but a 2.25 safety factor seemed very low. Braided hoses have safety factors between 3 and 4. Why is 2.25 acceptable in an expansion joint?

Flexible products are always riskier than solid pipe, so it seemed only right that our housed expansion joints and expansion compensators should have a **safety factor** between **3.5** and **4** as published and what we have worked to in all designs.

All expansion joints are more subject to collapse when fully extended. Our ratings are all at full extension. In many cases, the collapse may take place without leakage. However, when the pipe system cools down and the cycle reverses, the collapsed area is crushed and dramatic leakage follows. Should it turn out our spring rates are higher than some of our competitors, it is because of our safer bellows construction. Safety is far more important than spring rate.

We have an elaborate test facility to test Mercer Rubber Expansion Joints and all Stainless Steel products. All product designs are thoroughly tested before marketing. Unfortunately, our tests show design ratings by competitors are often optimistic. Testing rather than theoretical design is the only answer.

If you have never had a problem with an expansion compensator or housed expansion joint, everything we have discussed may seem unimportant. However, our improvements will keep both of us out of trouble and make life easier. Let's review.

MASON IMPROVEMENTS

- 1. Good operating clearances to prevent binding.
- 2. A much better locking mechanism that can be used to partially compress joints prior to installation to allow for contraction as well as expansion on installations with temperatures below ambient when in service. The locking device is a permanent part of the construction. They can always be tightened to hold the expansion joint in a particular position for removal while piping is serviced.
- 3. A depth gauge on the moving end to monitor movements, make certain that joints are at the 0 position before installation, or used to pre-set joints in some preferred position.
- 4. Great attention to canister and pipe welding details.
- 5. Internal and external clearances to prevent bellows wear because of rubbing.
- 6. Safety factors between 3.5 and 4, compared to others at 2.25. Double drains on most designs, generous lifting near the center of gravity for easier handling.
- 7. A raised face flange and a floating flange on all flanged products for easier installation and bellows anti-torquing protection.
- 8. Grooved fittings are beveled as well to allow welding into pipe lines as an alternate method. (Mason does not recommend welding.)
- 9. Our staff of in-house engineers holding licenses in virtually all of the states with the capability to design or review complete piping systems when our clients need those services. We will try to help you in any case, should you have bought from others that offer no engineering service. Seismic problems can be addressed as well.

If the following tables of externally pressurized stock items do not meet all of your requirements, please let us offer custom product. We also manufacture internally pressurized products and welcome inquiries on these as well. We are here to help.

Thanks for bearing with me.

Ann Mrain

Norm Mason





Spring Rate Test in Progress.

Failure



Competitor Pressure Failures.

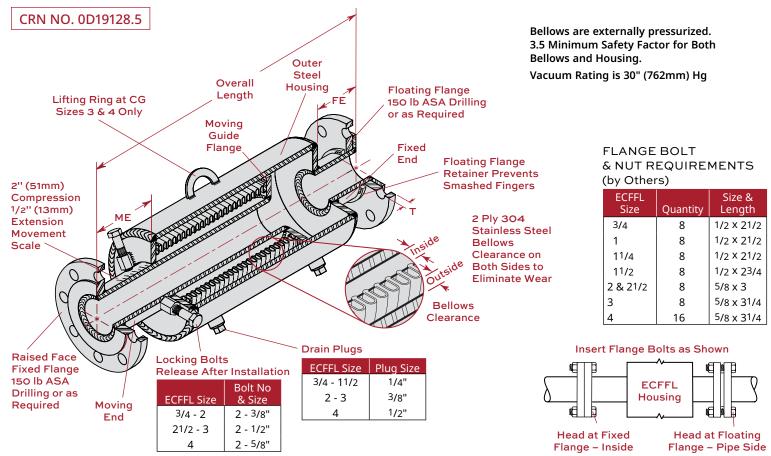


Hydro Pressure Test in Progress.

10.77



ECFFL- 2" (51mm) Movement Expansion Compensator with Fixed & Floating Flanges



TYPE ECFFL DIMENSIONS & PRESSURE RATINGS 2" (51mm) COMPRESSION, 1/2" (13mm) EXTENSION

	Pi	pe	Ove	erall		loving leutral		E End		iter ising	N	lominal Clear	l Bellov ance	VS	Sp	ring	-	st† @ I psi		ated ssure	Sh	nip
T	Si	ze	Len	gth	Len	ngth	Ler	ngth	0	.D.		side		side	Ŕ	ate	(13.8	bar)	@70°	F (21°C)	Wei	ight
Туре	(in)	(mm)	(in)	(mm)	(in)	(mm)	(in)	(mm)	(in)	(mm)	(in)	(mm)	(in)	(mm)	(lb/in)	(kg/cm)	(lb)	(kg)	(psi)	(kg/cm ²)	(ID)	(kg)
	3/4	20	121/2	318	31/2	89	13/4	44	27/8	73	0.10	3	0.43	11	89	16	350	159	200	14	11	5
	1	25	121/2	318	31/2	89	13/4	44	31/2	89	0.13	3	0.55	14	95	17	500	227	200	14	14	6
	11/4	32	13	330	33/4	95	2	51	4	102	0.15	4	0.47	12	103	18	800	363	200	14	15	7
	11/2	40	13	330	33/4	95	2	51	41/2	114	0.17	4	0.46	12	106	19	1100	499	200	14	19	9
ECFFL-	2	50	131/2	343	41/8	105	21/8	54	51/4	133	0.17	4	0.52	13	110	20	1600	726	200	14	24	11
	21/2	65	141/4	362	41/4	108	21/4	55	61/4	159	0.24	6	0.53	14	126	23	2400	1089	200	14	35	16
	3	80	143/4	375	41/2	115	21/2	65	65/8	168	0.32	8	0.37	9	140	25	3500	1588	200	14	47	21
	4	100	143/4	375	41/2	115	21/2	65	85/8	219	0.33	8	0.81	21	150	27	5200	2359	200	14	70	32

[†]Lower Thrust Forces in proportion at lower pressures, i.e. 100 psi Force = 100/225 x published Thrust. Forces on Pipe Anchors must include Thrust Force and Spring Force. Spring Force is determined by multiplying the joint Spring Rate by its Thermal Movement (in/mm).

RATED PRESSURES @ ELEVATED TEMPERATURES

Tempe	rature		ated ssure
(°F)	(°C)	(psi)	(kg/cm ²)
200	93	182	12.8
250	121	176	12.4
300	149	170	12.0
400	204	156	11.0
500	260	154	10.8
600	316	142	10.7
700	371	148	10.4
800	427	Not Rec	ommended

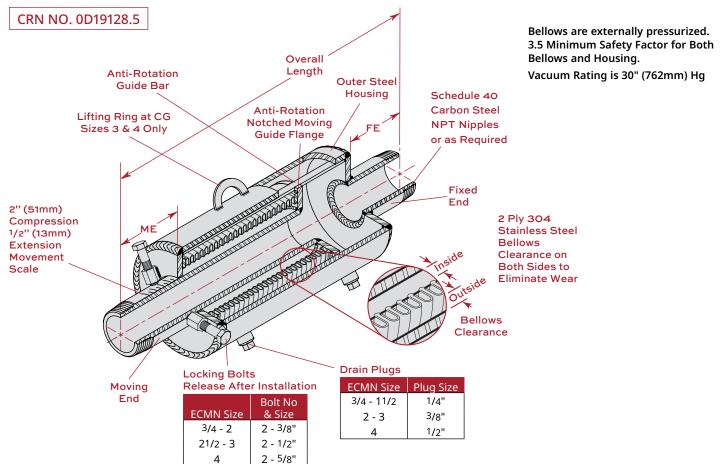
CARBON STEEL PLATE FLANGE THICKNESS

Pip	e Size		nge ness T
(in)	(mm)	(in)	(mm)
3/4 thru 4	20 thru 100	5/8	16

See Form A-36341-1, page 10.85, for Guide Spacing, Installation Instructions, Storage, and Inspection Criteria



ECMN- 2" (51mm) Movement Expansion Compensator with Threaded Nipples



TYPE ECMN DIMENSIONS & PRESSURE RATINGS 2" (51mm) COMPRESSION, 1/2" (13mm) EXTENSION

	Pi	pe	Ove	rall	ME M End N	oving eutral	F Fixec	E I Fnd		ter sing	N	ominal Clear		/S	Spr	ring		st†@ psi		ated ssure	Sh	ip
Туре		ze (mm)	Len (in)		Len (in)			gth (mm)	0. (in)		Ins (in)	ide (mm)	Out (in)	side (mm)	Ŕa	ate (kg/cm)		bar) (kg)	@70°	F (21°C) (kg/cm ²)	Wei	ight (kg)
	3/4	20	121/2	318	33/4	95	15/8	40	27/8	73	0.10	3	0.43	11	89	16	350	159	200	14	7	3
	1	25	121/2	318	33/4	95	15/8	40	31/2	89	0.13	3	0.55	14	95	17	500	227	200	14	10	4
	11/4	32	13	330	4	100	17/8	46	4	102	0.15	4	0.47	12	103	18	800	363	200	14	11	5
ECMN-	11/2	40	13	330	4	100	17/8	46	41/2	114	0.17	4	0.46	12	106	19	1100	499	200	14	13	6
ECIVIIN-	2	50	131/2	343	41/8	103	21/8	53	51/4	133	0.17	4	0.52	13	110	20	1600	726	200	14	16	7
	21/2	65	141/4	362	43/8	110	21/4	55	61/4	159	0.24	6	0.53	14	126	23	2400	1089	200	14	23	10
	3	80	143/4	375	41/2	115	21/2	65	65/8	168	0.32	8	0.37	9	140	25	3500	1588	200	14	32	15
	4	100	143/4	375	41/2	115	21/2	65	85/8	219	0.33	8	0.81	21	150	27	5200	2359	200	14	50	23

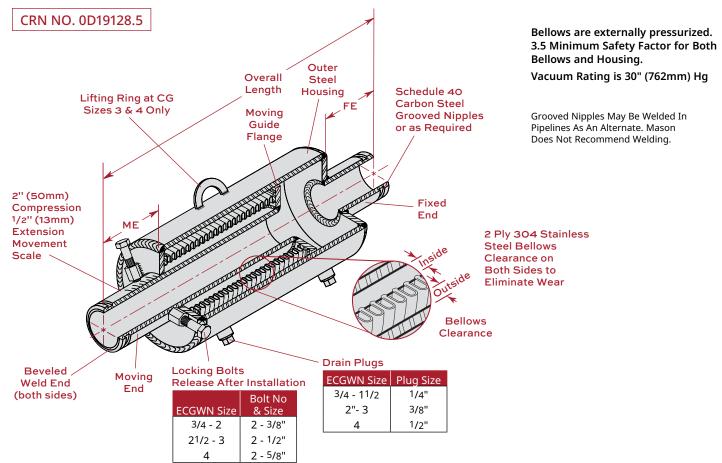
^tLower Thrust Forces in proportion at lower pressures, i.e. 100 psi Force = 100/225 x published Thrust. Forces on Pipe Anchors must include Thrust Force and Spring Force. Spring Force is determined by multiplying the joint Spring Rate by its Thermal Movement (in/mm).

RATED PRESSURES @ ELEVATED TEMPERATURES

Tempe	rature		ated ssure
(°F)	(°C)	(psi)	(kg/cm ²)
200	93	182	12.8
250	121	176	12.4
300	149	170	12.0
400	204	156	11.0
500	260	154	10.8
600	316	142	10.7
700	371	148	10.4
800	427	Not Rec	ommended

See Form A-36341-1, page 10.85, for Guide Spacing, Installation Instructions, Storage, and Inspection Criteria

ECGWN- 2" (51mm) Movement Expansion Compensator with Grooved Weld Nipples



TYPE ECGWN DIMENSIONS & PRESSURE RATINGS 2" (51mm) COMPRESSION, 1/2" (13mm) EXTENSION

	Pi	pe	Ove	rall	ME M End N	loving		E End		iter ising	N	lomina Cleai	l Bellov rance	VS	Sn	ring		st† @ I psi		ated	Sh	nip
Туре		ze (mm)	Len (in)	gth (mm)	Len (in)	igth (mm)	Ler (in)	ngth (mm)	0. (in)		Ins (in)	ide (mm)	Out (in)	side (mm)		ate		bar) (kg)	@70°	F (21°C) (kg/cm ²)	Wei	ight (kg)
	3/4	20	121/2	318	33/4	95	15/8	40	27/8	73	0.10	3	0.43	11	89	16	350	159	200	14	7	3
	1	25	121/2	318	33/4	95	15/8	40	31/2	89	0.13	3	0.55	14	95	17	500	227	200	14	9	4
	11/4	32	13	330	4	100	17/8	46	4	102	0.15	4	0.47	12	103	18	800	363	200	14	10	5
ECGWN-	11/2	40	13	330	4	100	17/8	46	41/2	114	0.17	4	0.46	12	106	19	1100	499	200	14	13	6
ECGWIN-	2	50	131/2	343	41/8	103	21/4	55	51/4	133	0.17	4	0.52	13	110	20	1600	726	200	14	17	8
	21/2	65	141/4	362	43/8	110	21/4	55	61/4	159	0.24	6	0.53	14	126	23	2400	1089	200	14	24	11
	3	80	143/4	375	41/2	115	21/2	65	65/8	168	0.32	8	0.37	9	140	25	3500	1588	200	14	33	15
	4	100	143/4	375	41/2	115	21/2	65	85/8	219	0.33	8	0.81	21	150	27	5200	2359	200	14	50	23

[†]Lower Thrust Forces in proportion at lower pressures, i.e. 100 psi Force = 100/225 x published Thrust. Forces on Pipe Anchors must include Thrust Force and Spring Force. Spring Force is determined by multiplying the joint Spring Rate by its Thermal Movement (in/mm).

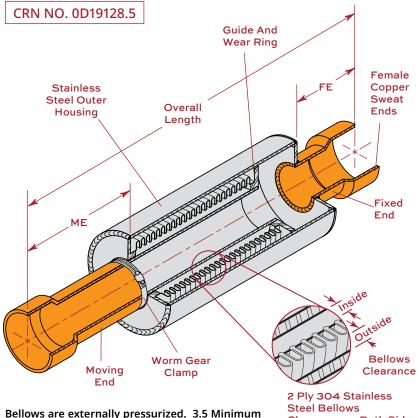
RATED PRESSURES @ ELEVATED TEMPERATURES

Tempe	rature		ated ssure
(°F)	(°C)	(psi)	(kg/cm ²)
200	93	182	12.8
250	121	176	12.4
300	149	170	12.0
400	204	156	11.0
500	260	154	10.8
600	316	142	10.7
700	371	148	10.4
800	427	Not Rec	ommended

See Form A-36341-1, page 10.85, for Guide Spacing, Installation Instructions, Storage, and Inspection Criteria



ECCPS- 2" (51mm) Movement Expansion Compensator with Copper Sweat Ends



Bellows are externally pressurized. 3.5 Minimum Safety Factor for Both Bellows and Housing. Vacuum Rating is 30" (762mm) Hg

Clearance on Both Sides to Eliminate Wear

ALSO AVAILABLE:

ECCPS-NSF

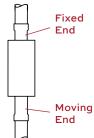
for Drinking Water Systems

LEAD FREE: The surface contacted by consumable water contains less than one quarter of one percent (0.25%) of lead by weight. These flexible joint fitting assemblies are UL Listed under File MH48651 and are intended for installation in accordance with the Mason installation instructions and the applicable requirements in Annex G of ANSI/NSF-61 and NSF-372.



WARNING! If disinfecting (chlorinating) is required per the International Plumbing Code, AWWA C651, and AWWA C652, then tablets and granular chlorine (calcium hypochlorite), and chlorine for swimming pool disinfection CANNOT be used on our products. We recommend chlorinating with diluted liquid chlorine (sodium hypochlorite) and immediately flushing thoroughly with potable water, as defined in the above Code.

All traces of chlorine must be removed, since residual chlorine will cause corrosion and lead to premature failure of our products. Failure to do so will void our warranty. Mason recommends installing hoses vertically where feasible to promote drainage of chlorine.



PRESSURE

Tempe	rature	Pre	ssure
(°F)	(°C)	(psi)	(kg/cm ²)
150	66	160	11
300	149	145	10
400	204	135	9

Rated

REDUCTION TABLE

TYPE ECCPS DIMENSIONS & PRESSURE RATINGS 2" (51mm) COMPRESSION, 1/2" (13mm) EXTENSION

		oing ze ^{tt}	Ove Len		End N	loving leutral lath	Fixed	E d End ath	Ηοι	ıter ısing .D.		lominal Clear ide	ance	vs side		ring ate	200	st† @ psi bar)	Pre	ated ssure F (21°C)	Sh Wei	ip aht
Туре	(in)	(mm)	(in)	(mm)	(in)	(mm)	(in)	(mm)	(in)	(mm)	(in)	(mm)	(in)	(mm)		(kg/cm)	(lb)	(kg)		(kg/cm ²)		(kg)
	3/4	20	111/2	292	31/8	79	15/8	40	2	51	0.17	4	0.11	3	23	4	320	145	200	14	2	1
	1	25	111/2	292	31/8	79	15/8	40	2	51	0.22	6	0.13	3	44	8	520	236	200	14	2	1
	11/4	32	12	305	31/2	89	13/4	44	23/4	70	0.20	5	0.22	6	50	9	630	286	200	14	3	2
FCCDC	11/2	40	12	305	31/2	89	13/4	44	23/4	70	0.17	4	0.20	5	98	18	750	340	200	14	4	2
ECCPS-	2	50	121/4	311	33/4	95	13/4	44	31/2	89	0.16	4	0.13	3	168	30	1160	526	200	14	5	2
	21/2	65	131/4	337	41/4	108	21/8	54	4	102	0.20	5	0.22	6	195	35	1810	821	200	14	6	3
	3	80	131/4	337	41/4	108	21/8	54	41/4	114	0.21	5	0.28	7	316	57	2440	1107	200	14	7	3
	4	100	141/2	368	43/8	111	21/2	65	6	150	0.14	4	0.30	8	350	63	3700	1678	200	14	25	11

[†]Lower Thrust Forces in proportion at lower pressures, i.e. 100 psi Force = 100/225 x published Thrust. Forces on Pipe Anchors must include Thrust Force and Spring Force. Spring Force is determined by multiplying the joint Spring Rate by its Thermal Movement (in/mm).

⁺⁺Female end fits over copper tubing, e.g. 1¹/2" (40mm) fits over 1¹/2" (40mm) tubing.

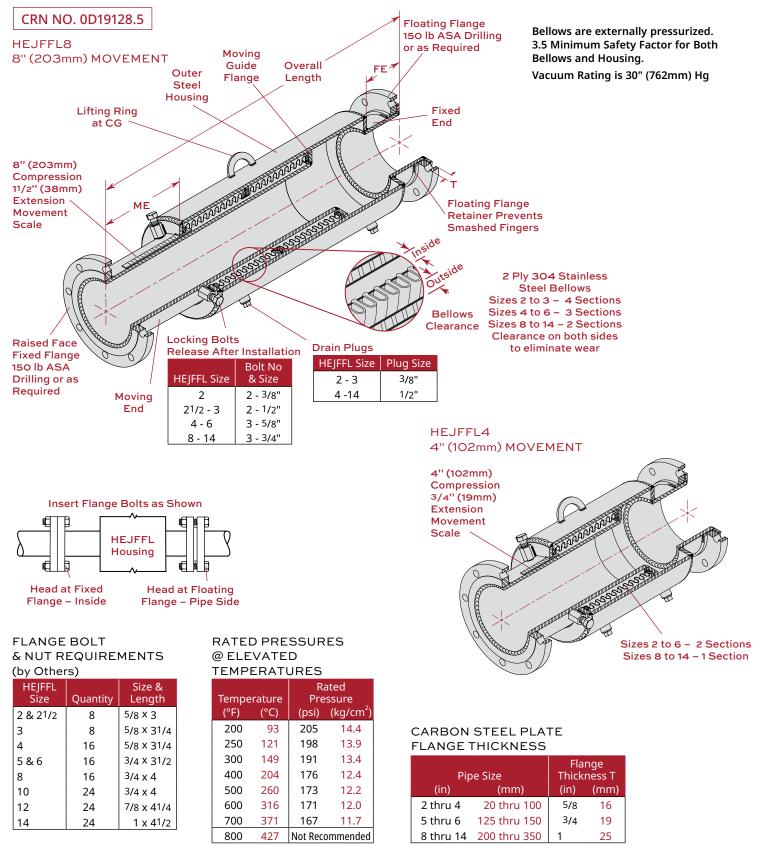
INSTALLATION

- 1. Thoroughly clean male and female ends using steel wool and steel brushes.
- 2. Apply flux.
- 3. Heat joint for proper flow of silver solder. Silver solder flows around 430°F. Composition is silver and tin only. There should be no lead content.
- 4. Do not use brazing rod or other high temperature techniques. Overheating will cause leaks.
- 5. Remove Worm Gear Clamp.

See Form A-36341-1, page 10.85, for Guide Spacing, Installation Instructions, Storage, and Inspection Criteria



HEJFFL- Housed Expansion Joint with Fixed & Floating Flanges





TYPE HEJFFL4 DIMENSIONS & PRESSURE RATINGS 4" (102mm) COMPRESSION, 3/4" (19mm) EXTENSION

	Pir)e	Ove	rall	ME M End N			E l End	Out Hous		N		l Bellov ance	vs	Spr	ing	Thru: 225			ated ssure	Sh	nip
-	Siz	e	Len	gth	Len	gth	Len	gth	0.0). Č		ide	Out		Ra	ite	(15.5	bar)	@70°	F (21°C)	Wei	ight
Туре	(in)	(mm)	(in)	(mm)	(in)	(mm)	(in)	(mm)	(in)	(mm)	(in)	(mm)	(in)	(mm)	(lb/in)	(kg/cm)	(lb)	(kg)	(psi)	(kg/cm ²)	(lb)	(kg)
	2	50	261/2	673	83/8	213	31/8	79	59/16	141	0.39	10	0.39	10	165	30	2500	1134	225	16	50	23
	21/2	65	261/2	673	83/8	213	31/8	79	59/16	141	0.25	6	0.25	6	235	42	2700	1225	225	16	51	23
	3	80	261/2	673	83/8	213	31/8	79	65/8	168	0.32	8	0.33	8	240	43	3900	1769	225	16	65	29
	4	100	261/2	673	83/8	213	31/8	79	85/8	219	0.49	12	0.35	9	300	54	6900	3130	225	16	87	39
	5	125	281/2	723	83/8	213	41/8	105	95/8	244	0.39	10	0.41	10	400	72	9500	4309	225	16	90	41
HEJFFL4-	6	150	281/2	723	83/8	213	41/8	105	103/4	273	0.39	10	0.39	10	500	90	12600	5715	225	16	137	62
	8	200	281/2	723	83/8	213	41/8	105	123/4	384	0.39	10	0.39	10	600	107	19000	8618	225	16	180	82
	10	250	281/2	723	83/8	213	41/8	105	16	406	0.53	13	0.53	13	800	143	30000	13608	225	16	230	104
	12	300	281/2	723	83/8	213	41/8	105	18	457	0.42	11	0.41	10	1175	210	40000	18144	225	16	273	124
	14	350	30	762	83/8	213	41/8	105	20	500	0.43	11	0.42	11	1400	250	64000	29030	225	16	320	145

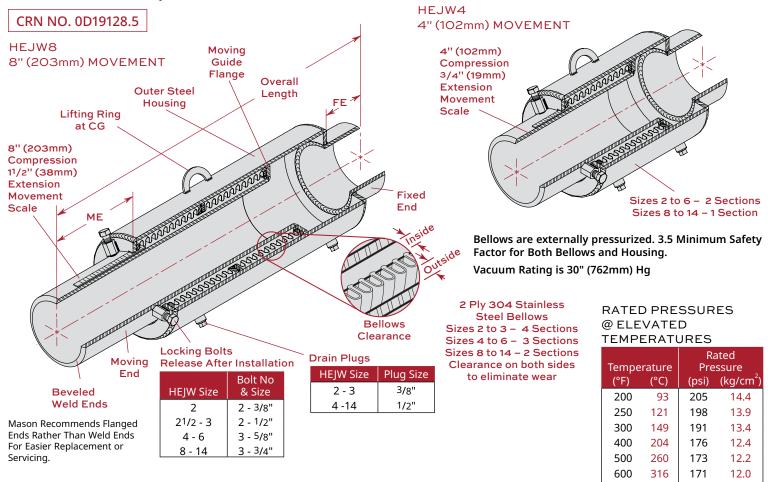
TYPE HEJFFL8 DIMENSIONS & PRESSURE RATINGS 8" (203mm) COMPRESSION, 11/2" (38mm) EXTENSION

	Pir)e	Ov	erall	ME M		F Fixed		Out Hous		N	ominal Clear	Bellov ance	VS	Spi	ring	Thru 225	st† @ psi		ated ssure	Sh	nip
Туре	Siz (in)			ngth (mm)	Len (in)		Len (in)		0.[(in)	J	Ins (in)	ide (mm)	Out: (in)		Ŕa	ate (kg/cm)	(15.5	bar) (kg)	@70°	F (21°C) (kg/cm ²)	Wei	ight (kg)
	2	50	43	1092	107/8	276	31/8	79	59/16	141	0.39	10	0.39	10	83	15	2500	1134	225	16	50	23
	21/2	65	43	1092	107/8	276	31/8	79	59/16	141	0.25	6	0.25	6	118	21	2700	1225	225	16	51	23
	3	80	43	1092	107/8	276	31/8	79	65/8	168	0.32	8	0.33	8	120	22	3900	1769	225	16	65	29
	4	100	43	1092	107/8	276	31/8	79	85/8	219	0.49	12	0.35	9	150	27	6900	3130	225	16	87	39
	5	125	47	1194	127/8	327	41/8	105	95/8	244	0.39	10	0.41	10	200	36	9500	4309	225	16	90	41
HEJFFL8-	6	150	47	1194	127/8	327	41/8	105	103/4	273	0.39	10	0.39	10	250	45	12600	5715	225	16	137	62
	8	200	47	1194	127/8	327	41/8	105	123/4	384	0.39	10	0.39	10	300	54	19000	8618	225	16	180	82
	10	250	47	1194	127/8	327	41/8	105	16	406	0.53	13	0.53	13	400	72	30000	13608	225	16	230	104
	12	300	47	1194	127/8	327	41/8	105	18	457	0.42	11	0.41	10	588	105	40000	18144	225	16	273	124
	14	350	50	1270	127/8	327	41/8	105	20	500	0.43	11	0.42	11	700	125	64000	29030	225	16	320	145

^tLower Thrust Forces in proportion at lower pressures, i.e. 100 psi Force = 100/225 x published Thrust. Forces on Pipe Anchors must include Thrust Force and Spring Force. Spring Force is determined by multiplying the joint Spring Rate by its Thermal Movement (in/mm).



HEJW– Housed Expansion Joint with Weld Ends



See Form A-36341-1, page 10•85, for Guide Spacing, Installation Instructions, Storage, and Inspection Criteria

TYPE HEJW4 DIMENSIONS & PRESSURE RATINGS 4" (102mm) COMPRESSION, 3/4" (19mm) EXTENSION

	Pip		Ove Len		ME M End N Len	eutral	Fixec		Out Hous O.[ing	N Ins	Clear	Bellov ance Out		Spr	ing ite	225	st† @ psi bar)	Pres	ited ssure = (21°C)		nip ight
Туре	(in)	(mm)	(in)	(mm)	(in)	(mm)		(mm)	(in)). (mm)	(in)	(mm)	(in)	(mm)		(kg/cm)	N	(kg)		(kg/cm ²)		(kg)
	2	50	22	559	51/2	140	11/2	38	59/16	141	0.39	10	0.39	10	165	30	2500	1134	225	16	50	23
	21/2	65	22	559	51/2	140	11/2	38	59/16	141	0.25	6	0.25	6	235	42	2700	1225	225	16	51	23
	3	80	22	559	51/2	140	11/2	38	65/8	168	0.32	8	0.33	8	240	43	3900	1769	225	16	65	29
	4	100	241/2	622	6	152	11/2	38	85/8	219	0.49	12	0.35	9	300	54	6900	3130	225	16	87	39
	5	125	241/2	622	6	152	21/2	64	95/8	244	0.39	10	0.41	10	400	72	9500	4309	225	16	90	41
HEJW4-	6	150	241/2	622	6	152	21/2	64	103/4	273	0.39	10	0.39	10	500	90	12600	5715	225	16	137	62
	8	200	241/2	622	6	152	21/2	64	123/4	384	0.39	10	0.39	10	600	107	19000	8618	225	16	180	82
	10	250	241/2	622	6	152	21/2	64	16	406	0.53	13	0.53	13	800	143	30000	13608	225	16	230	104
	12	300	241/2	622	6	152	21/2	64	18	457	0.42	11	0.41	10	1175	210	40000	18144	225	16	273	124
	14	350	26	660	6	152	21/2	64	20	500	0.43	11	0.42	11	1400	250	64000	29030	225	16	320	145

TYPE HEJW8 DIMENSIONS & PRESSURE RATINGS 8" (203mm) COMPRESSION, 11/2" (38mm) EXTENSION

	Pipe Size (in) (mm)		Overall Length (in) (mm)		ME Moving End Neutral Length (in) (mm)				Outer Housing O.D. (in) (mm)		Nominal Bellows Clearance				Spring		Thrust ⁺ @ 225 psi		Rated Pressure		Ship	
Туре											Inside (in) (mm)		Outside (in) (mm)		Rate (lb/in) (kg/cm)		(15.5 bar)		@70°F (21°C) (psi) (kg/cm²)		Weight	
HEJW8-	2	50	40	1016	91/2	241	11/2	38	59/16	141	0.39	10	0.39	10	83	15	2500	1134	225	16	50	23
	21/2	65	40	1016	91/2	241	11/2	38	59/16	141	0.25	6	0.25	6	118	21	2700	1225	225	16	51	23
	3	80	40	1016	91/2	241	11/2	38	65/8	168	0.32	8	0.33	8	120	22	3900	1769	225	16	65	29
	4	100	40	1016	91/2	241	11/2	38	85/8	219	0.49	12	0.35	9	150	27	6900	3130	225	16	87	39
	5	125	421/2	1080	10	254	21/2	64	95/8	244	0.39	10	0.41	10	200	36	9500	4309	225	16	90	41
	6	150	421/2	1080	10	254	21/2	64	103/4	273	0.39	10	0.39	10	250	45	12600	5715	225	16	137	62
	8	200	421/2	1080	10	254	21/2	64	123/4	384	0.39	10	0.39	10	300	54	19000	8618	225	16	180	82
	10	250	421/2	1080	10	254	21/2	64	16	406	0.53	13	0.53	13	400	72	30000	13608	225	16	230	104
	12	300	421/2	1080	10	254	21/2	64	18	457	0.42	11	0.41	10	588	105	40000	18144	225	16	273	124
	14	350	451/2	1156	10	254	21/2	64	20	500	0.43	11	0.42	11	700	125	64000	29030	225	16	320	145

[†]Lower Thrust Forces in proportion at lower pressures, i.e. 100 psi Force = 100/225 x published Thrust. Forces on Pipe Anchors must include Thrust Force and Spring Force. Spring Force is determined by multiplying the joint Spring Rate by its Thermal Movement (in/mm).



700

800

371

427

167

11.7

Not Recommended